



UNIVERSITY OF LEEDS

TESI DI LAUREA MAGISTRALE INGEGNERIA MECCANICA

Goal driven multi-objective shape optimization for conjugate heat transfer in an effusion cooling system of a combustion chamber, through a CFD-mesh-morphing based approach.

Relatore

Prof. Marco E. Biancolini

Correlatore

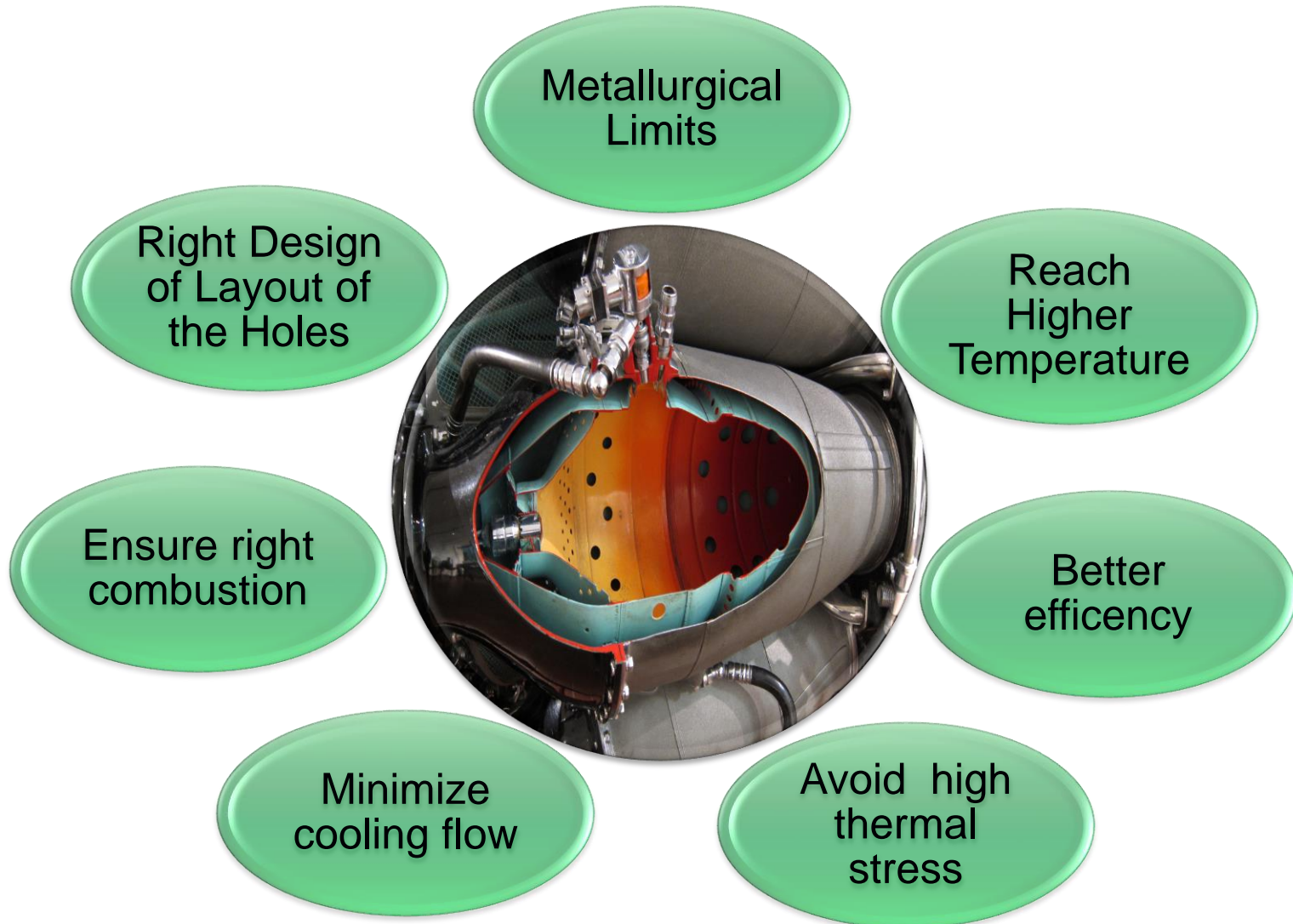
Prof. G.E. Andrews (University of Leeds)

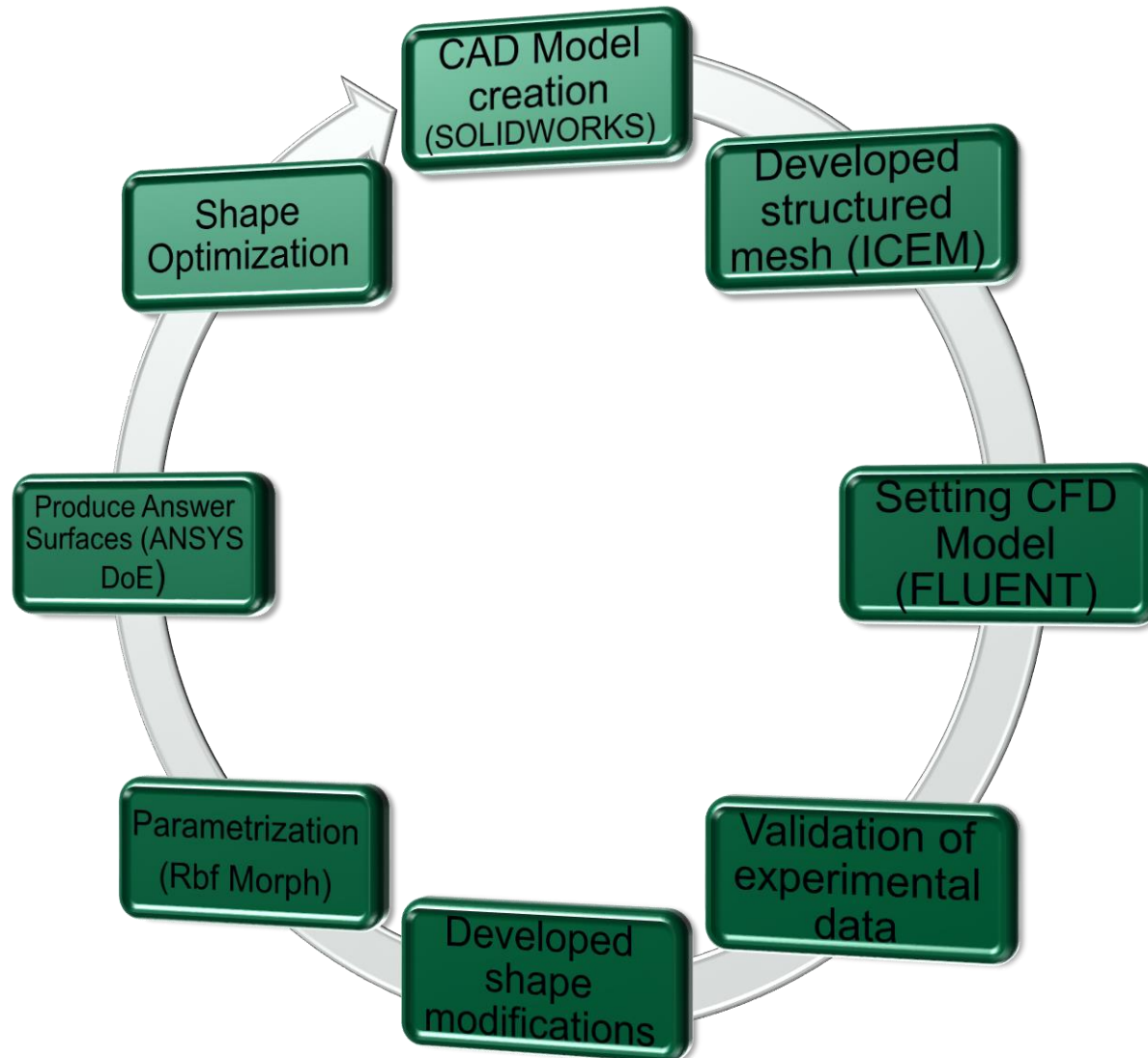
Ing. A. Pranzitelli (University of Leeds)

Laureando

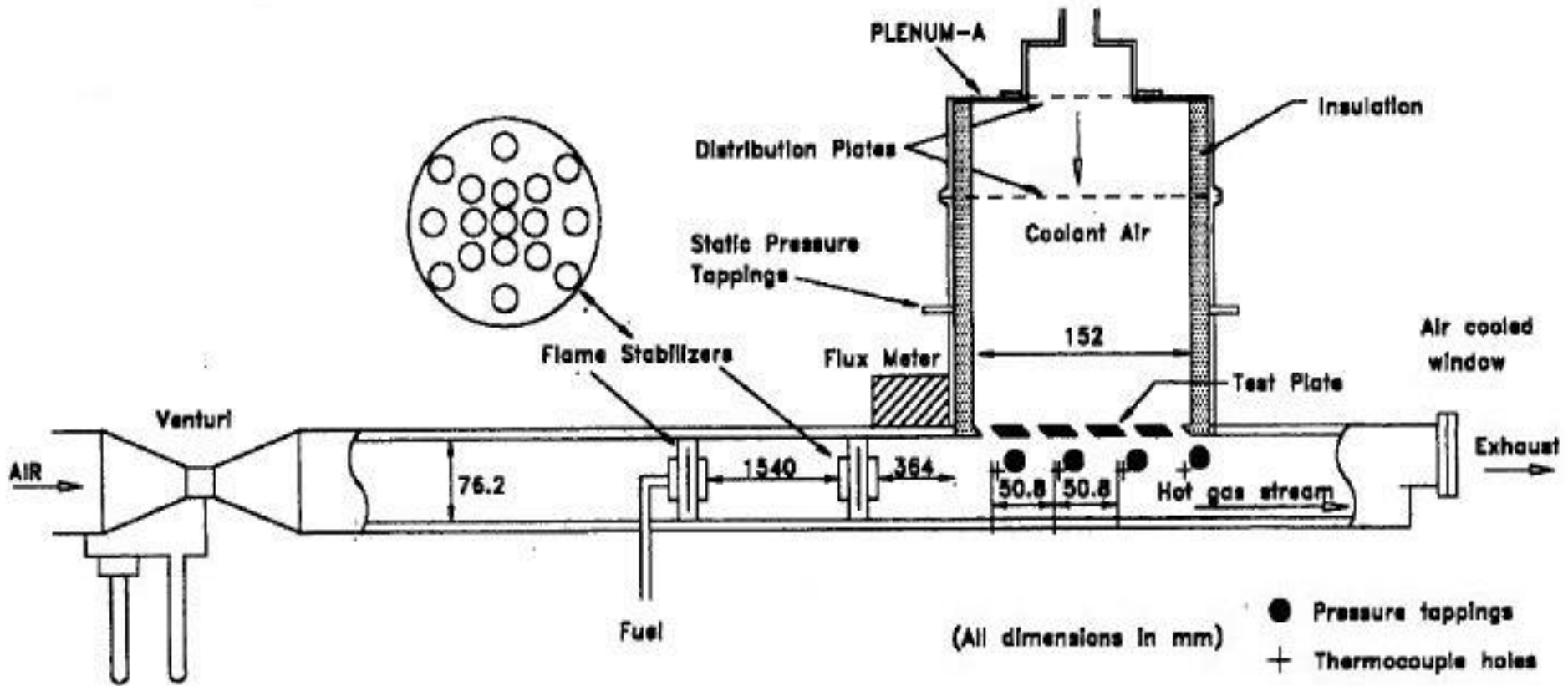
Walter Savastano

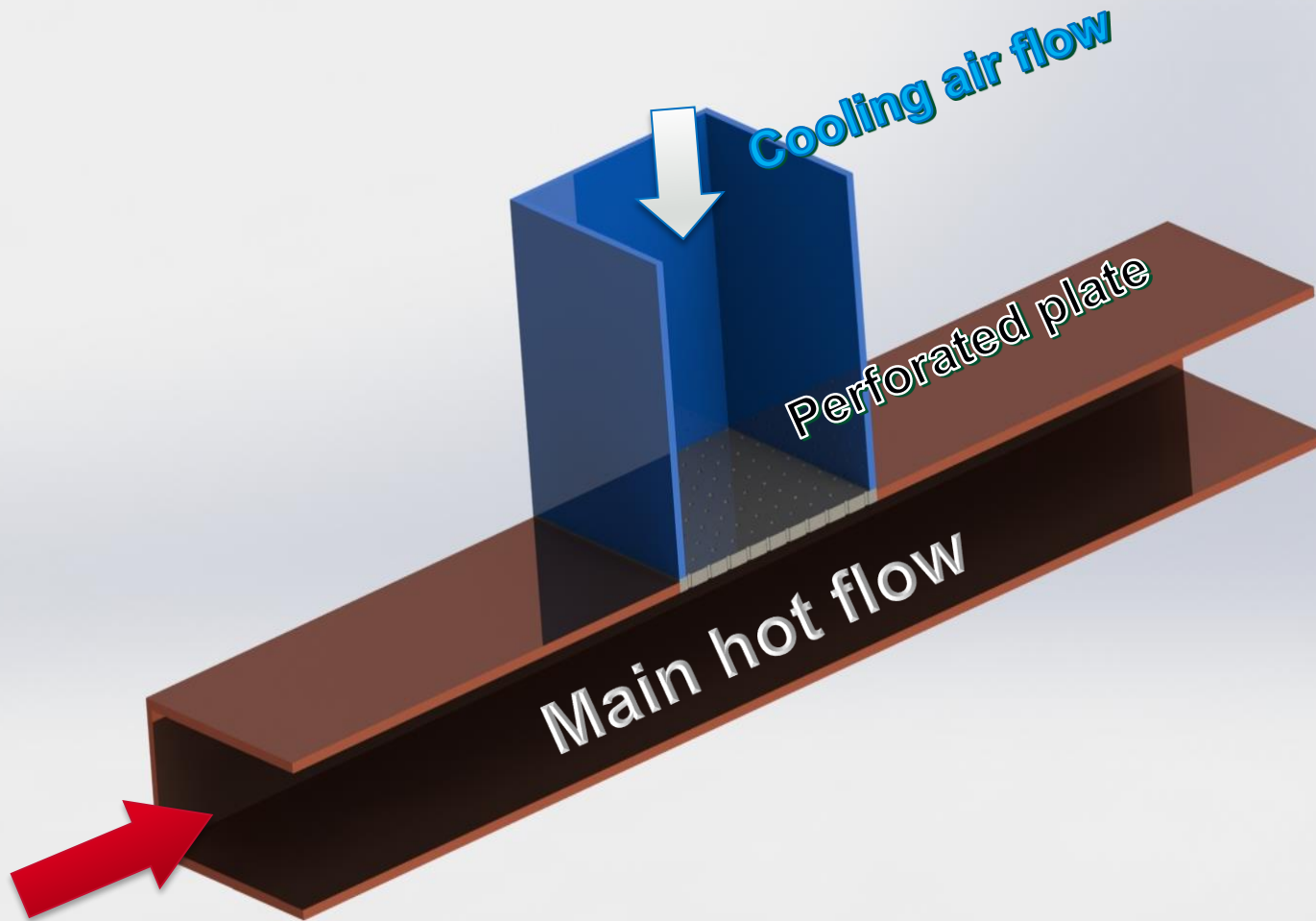
Matr. 0185986



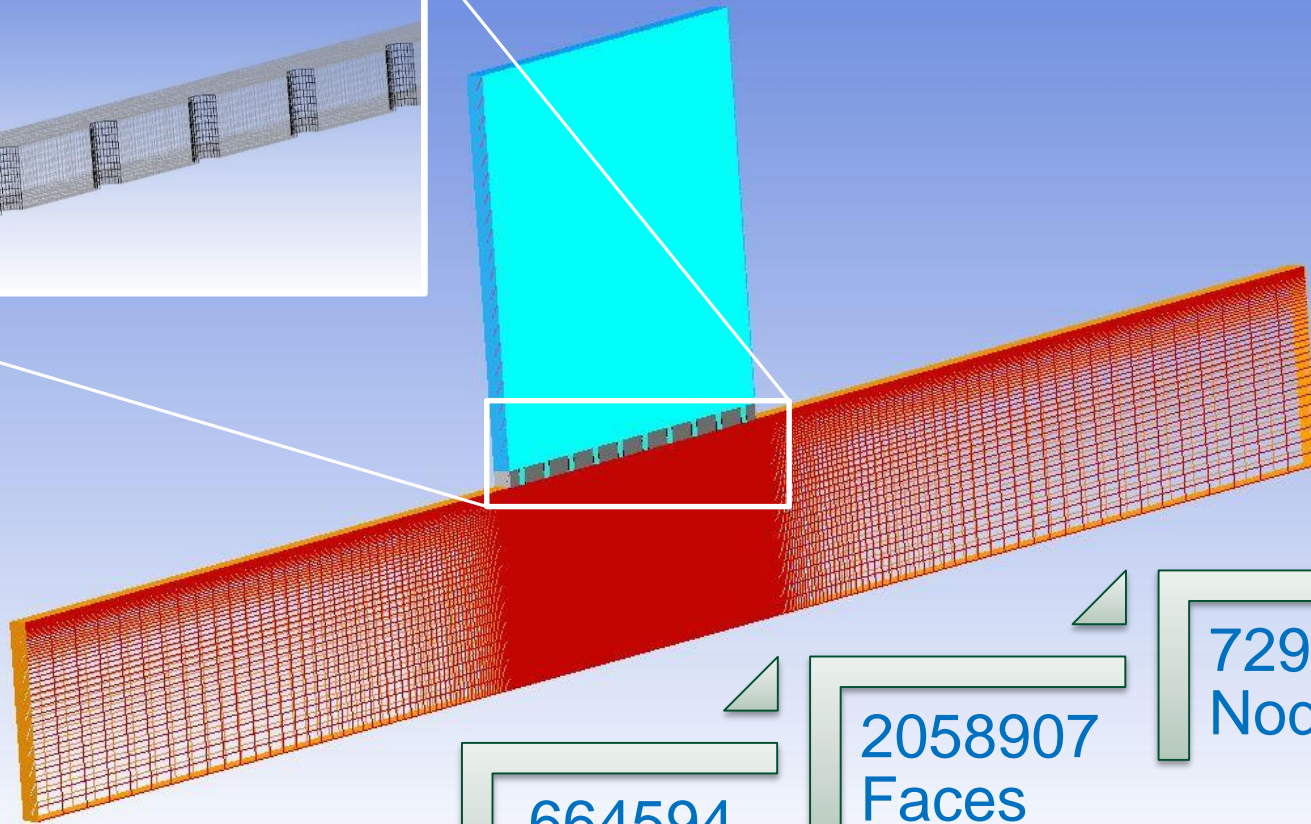
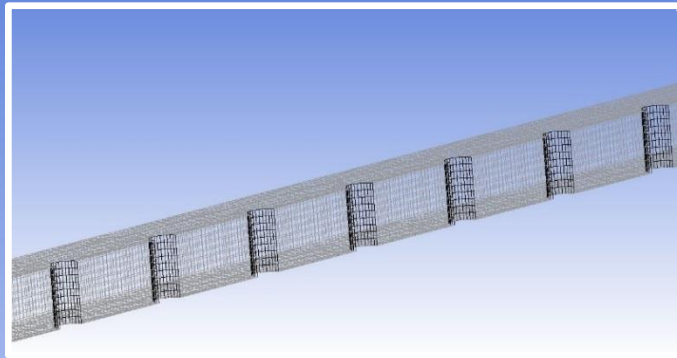


Experimental Apparatus





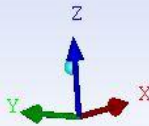
ANSYS ICEM: Structured mesh



664594
Cells

2058907
Faces

729388
Nodes



Fluent Model

Fluent

- Finite volume numerical calculation

Turbulence model

- Realizable k- ϵ

Wall Function

- Standard wall-function

Radiation Model

- Discrete Ordinate

Vel = 27 m/s

Temp = 770 K

Velocity inlet

Cold mass flow inlet

Mass flow rate
 $G = 0.4 \text{ kg/m}^2\text{s}$

$T = 300 \text{ K}$

NIMONIC
75

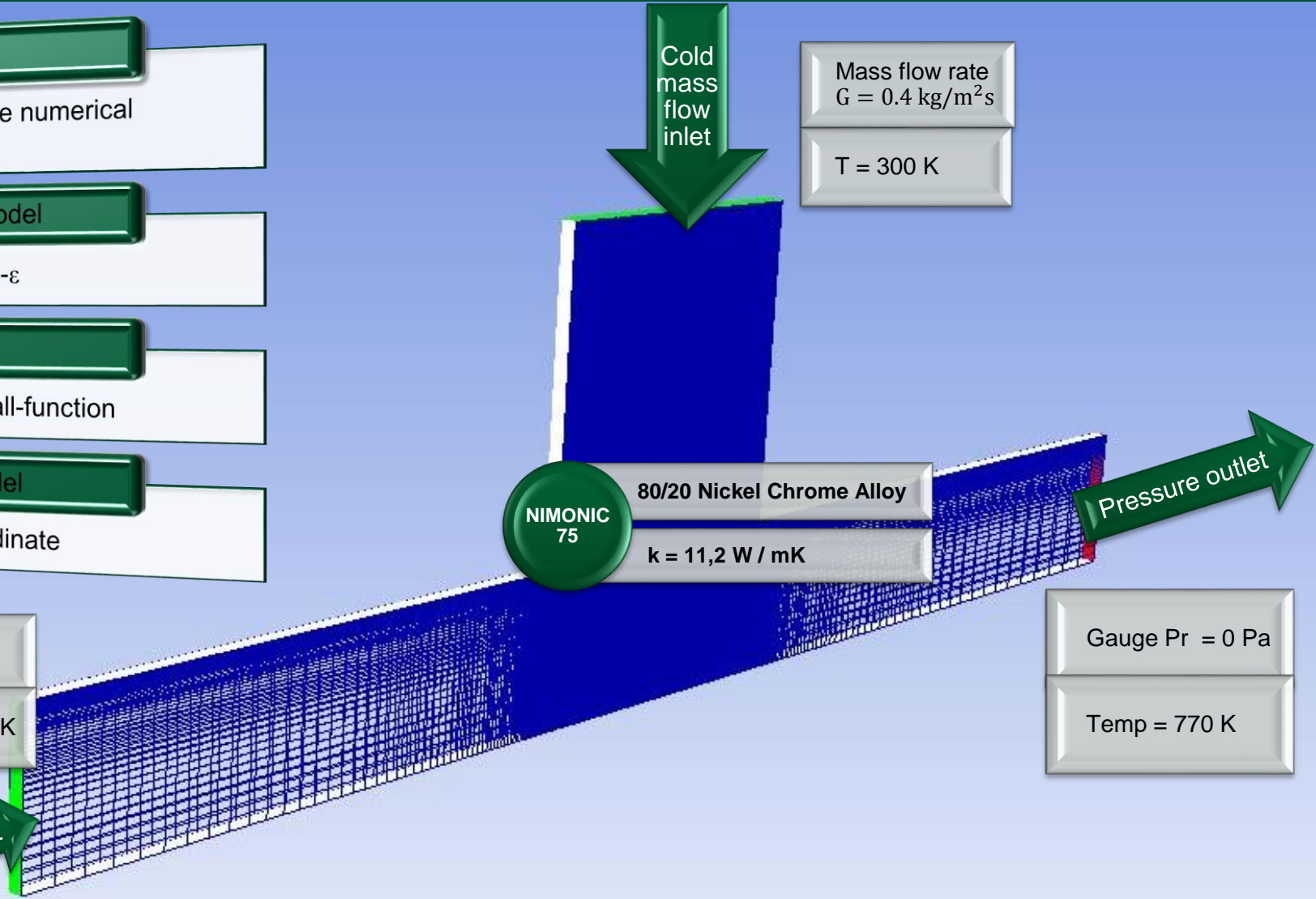
80/20 Nickel Chrome Alloy

$k = 11,2 \text{ W / mK}$

Pressure outlet

Gauge Pr = 0 Pa

Temp = 770 K



Overall effectiveness

$$\eta_{ov} = \frac{T_g - T_w}{T_g - T_c}$$

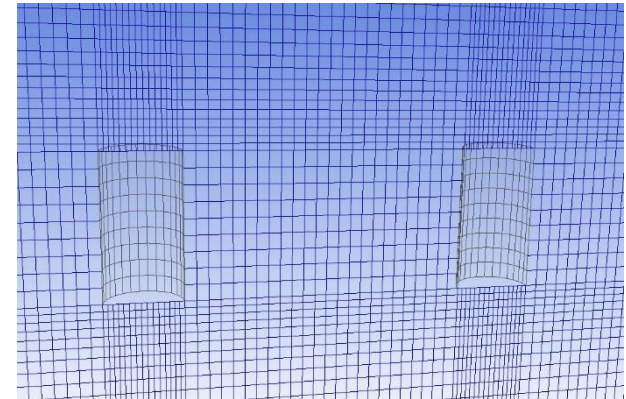
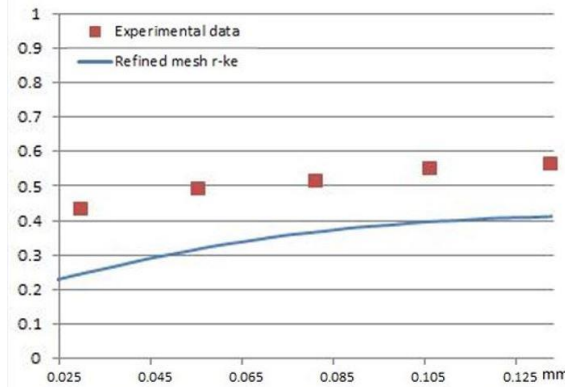
T_g = Hot gas Temperature

T_w = Wall temperature

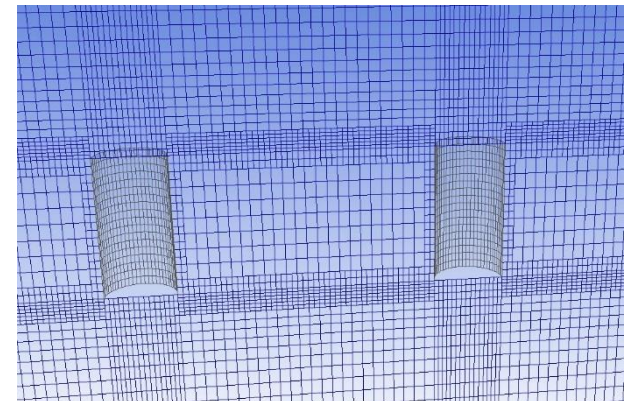
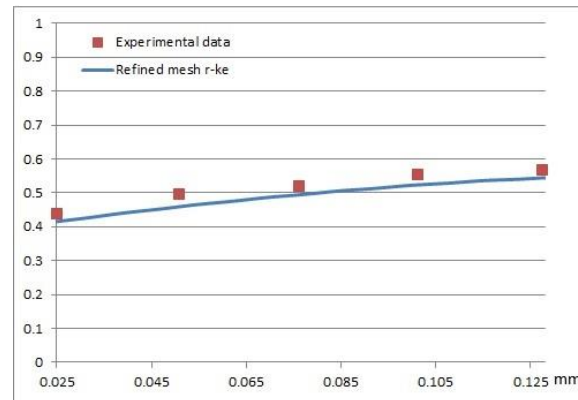
T_c = Cooling air Temperature



Coarse Mesh
(700k cells)



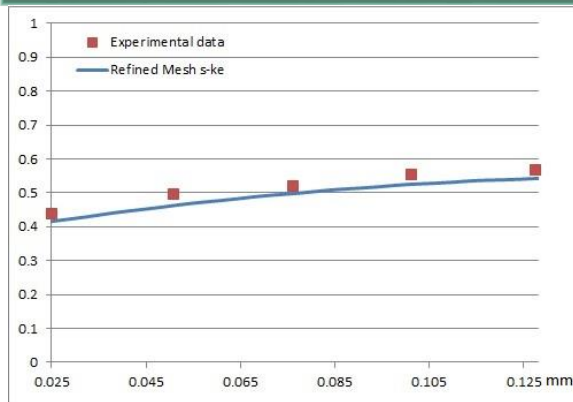
Refined Mesh
(1mln cells)



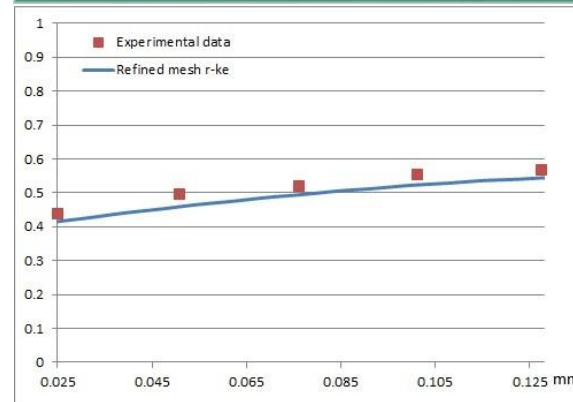
Turbulence Model Sensibility



Standard k-e



Realizable k-e



Overall effectiveness

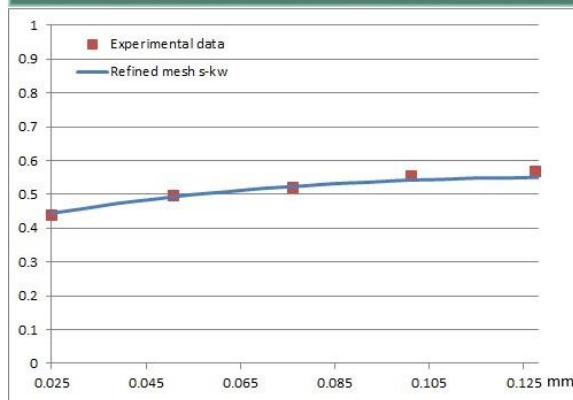
$$\eta_{ov} = \frac{T_g - T_w}{T_g - T_c}$$

T_g = Hot gas Temperature

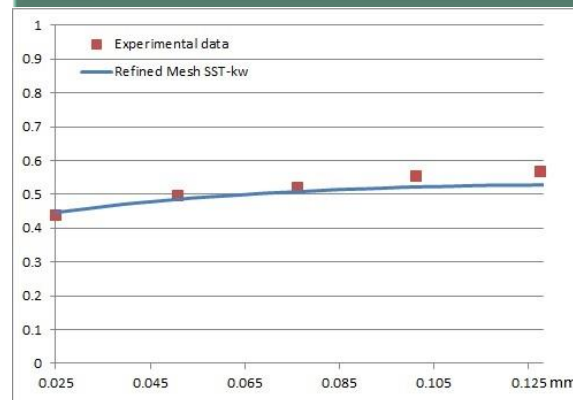
T_w = Wall temperature

T_c = Cooling air Temperature

Standard k-w



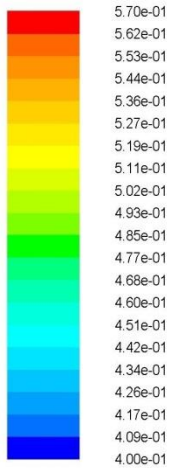
SST k-w



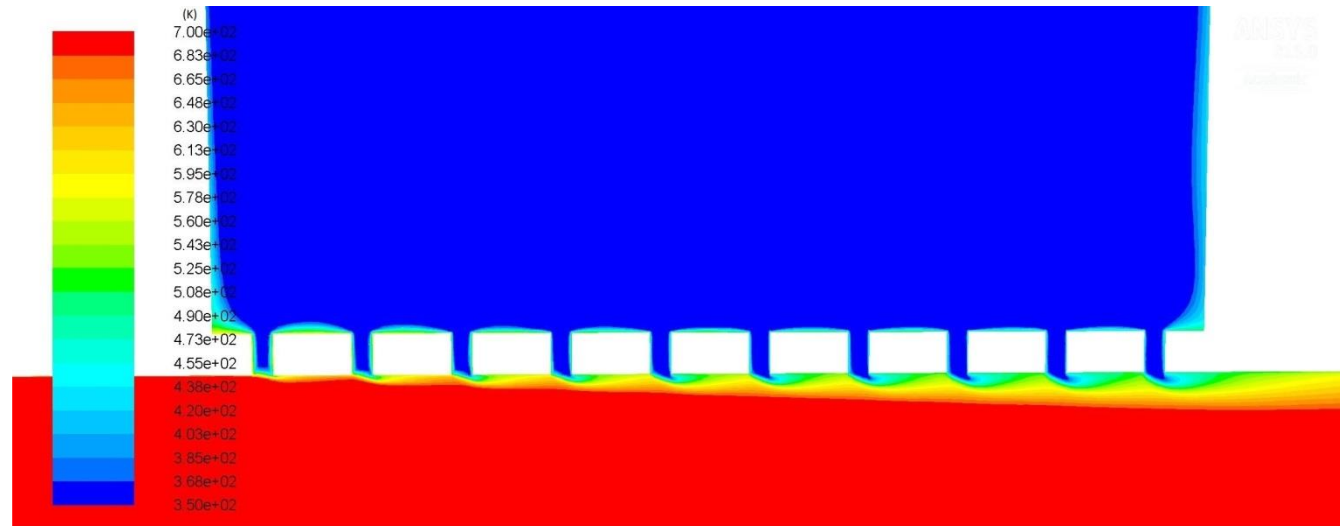
Baseline results

*Overall Effectiveness:
Contour on the plate*

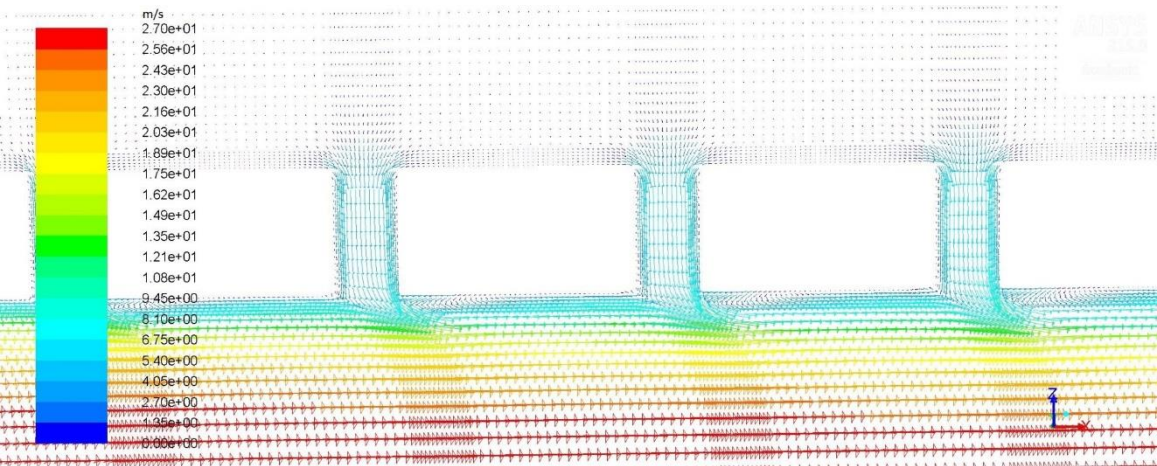
$$\eta_{ov} = \frac{T_g - T_w}{T_g - T_c}$$



*Temperature profile
on symmetry plane*



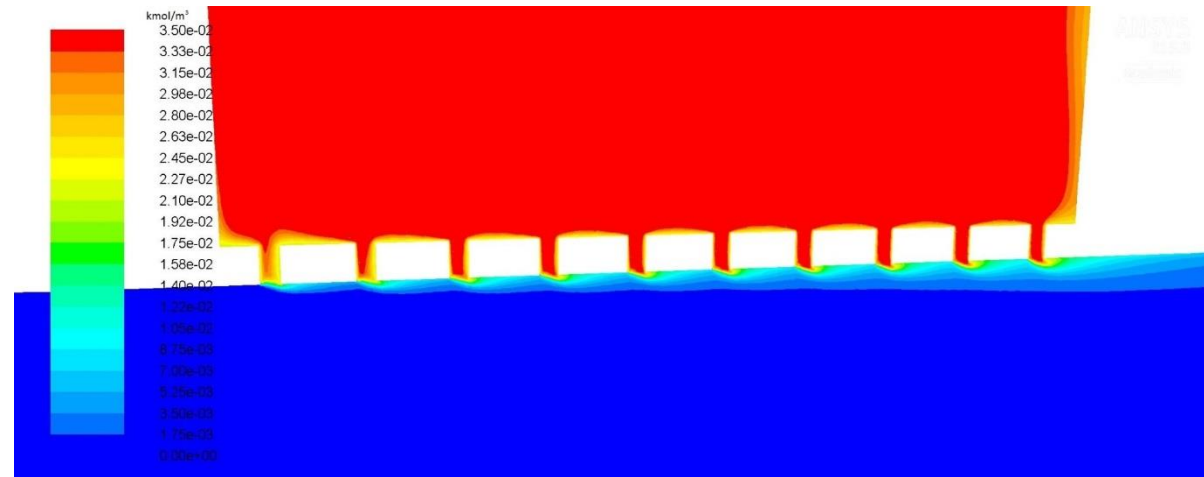
Velocity vectors on symmetry plate

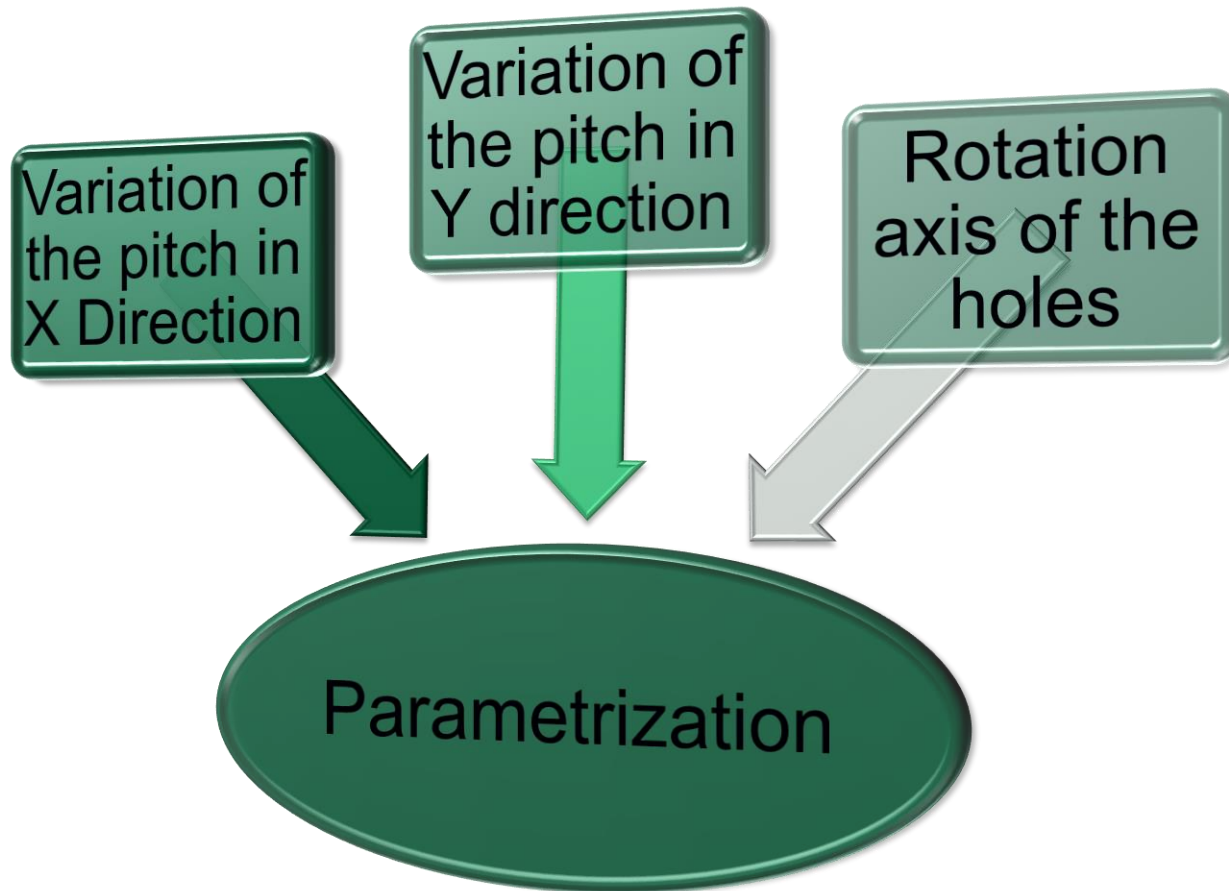


Flow separation at the exit of the holes

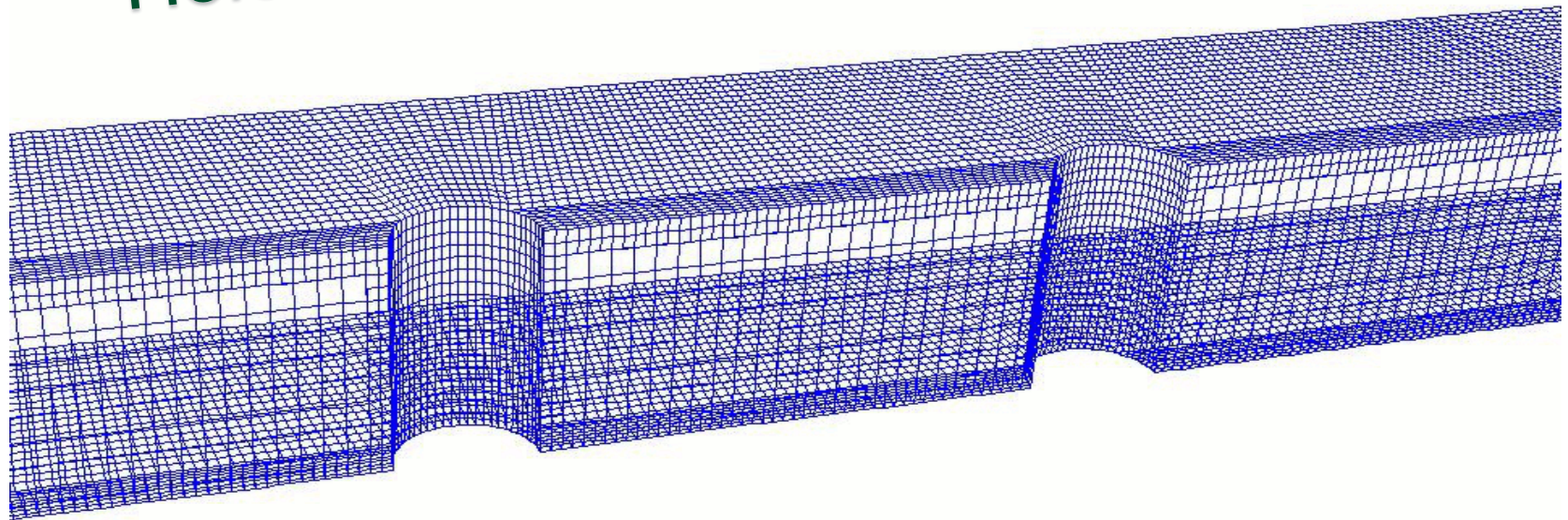
Tracer concentration on symmetry plane

No mix between the flows





Holes' Axis rotation

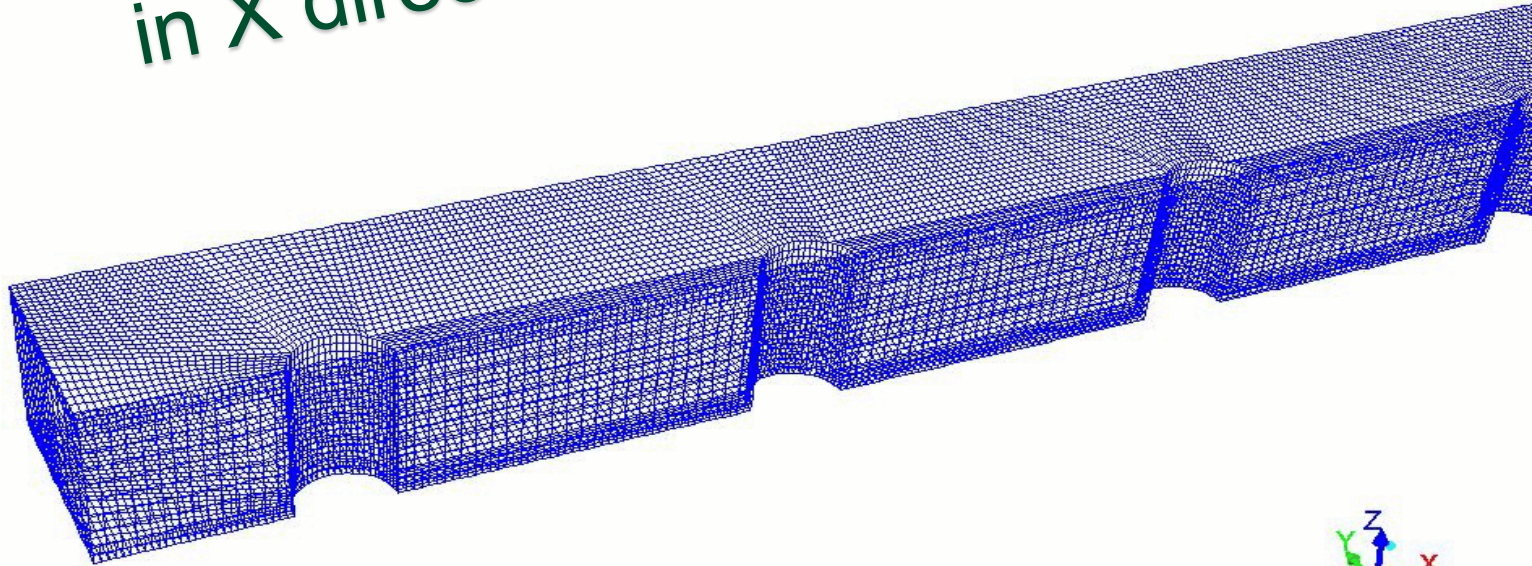


RBF Morph Parametrization

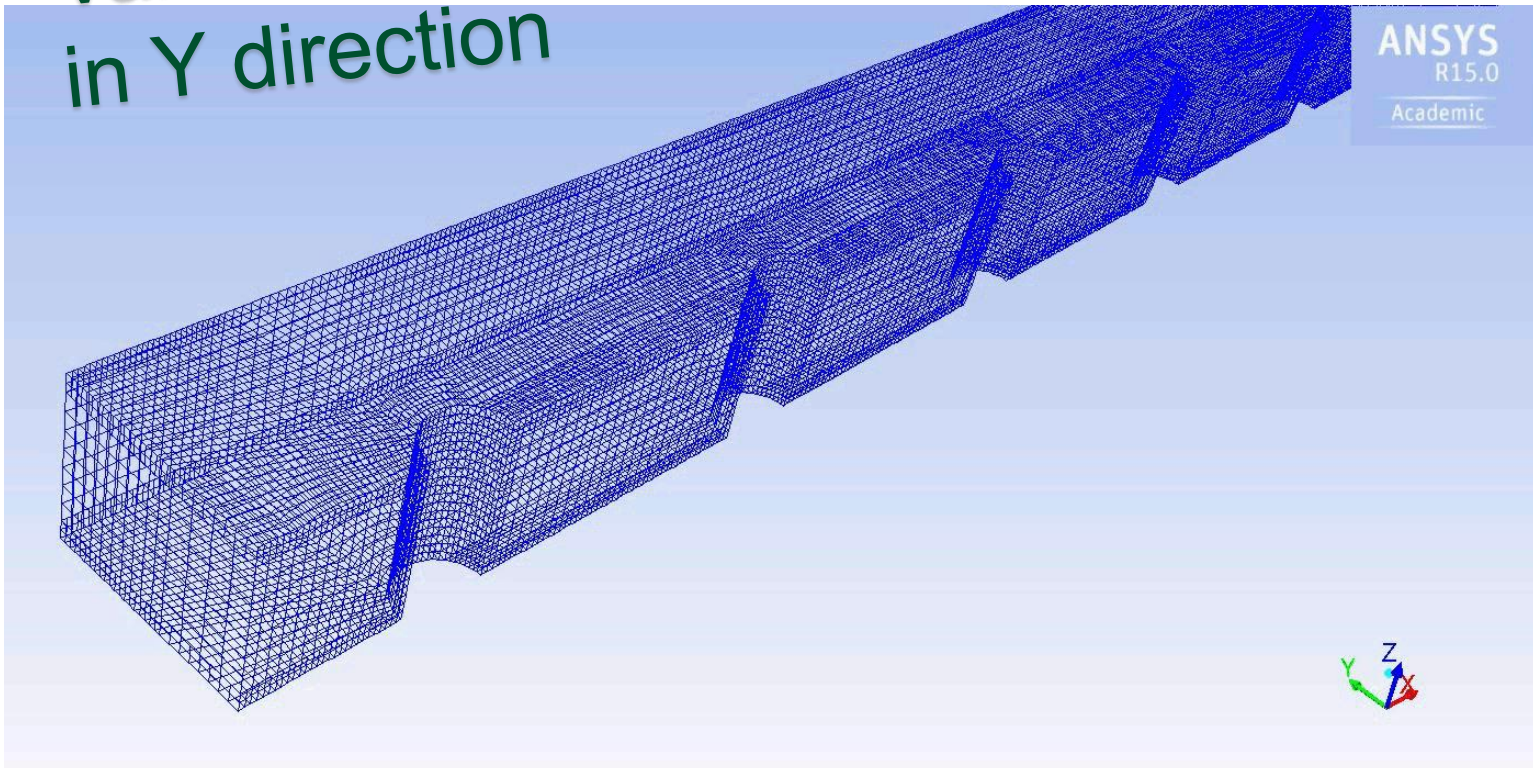


Variation of the pitch
in X direction

ANSYS
RUSH
APDL



Variation of the pitch
in Y direction



Input

- Rotation
- Pitch in X
- Pitch in Y

Output $\left(\frac{T_g - T_w}{T_g - T_c} \right)$

- **Overall effectiveness Average**
- Overall effectiveness Min **>0,4**
- Overall effectiveness Max
- Adiab. effectiveness Average (at 0,2 mm from the plate)
- Adiab. Effectiveness Max (at 0,2 mm from the plate)

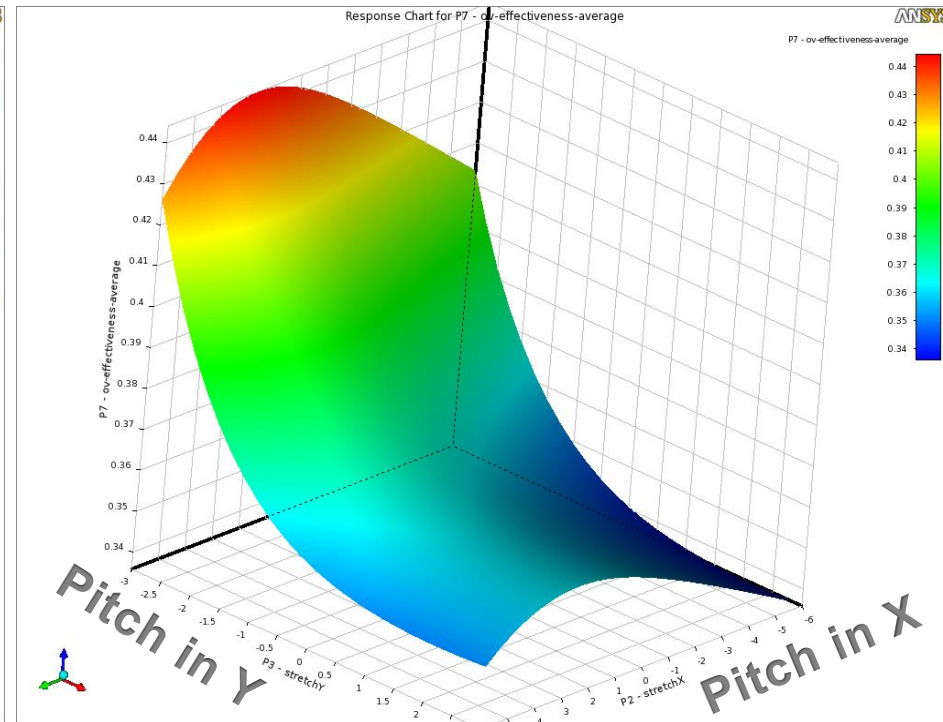
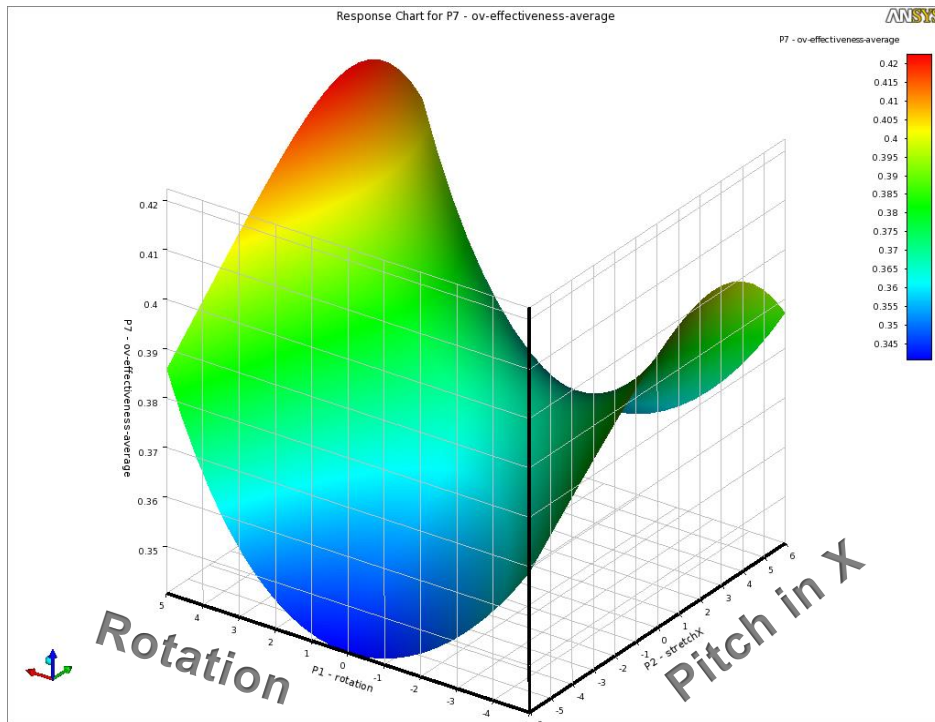
Input

Output

Injection angle (deg)	Pitch in x (mm)	Pitch in Y (mm)	Overall Effectiveness Min	Overall Effectiveness Average	Overall Effectiveness max	Adiabatic Effectiveness Average	Adiabatic Effectiveness Max
43°	18.8	19.5	0.408	0.468	0.616	0.243	0.356
90°	23.7	15.2	0.358	0.409	0.514	0.216	0.29
-51°	15.2	12.8	0.367	0.383	0.550	0.223	0.314
-43°	6.7	13.4	0.405	0.413	0.650	0.249	0.388
-78°	22.5	16.4	0.275	0.351	0.573	0.204	0.323
57°	10.3	18.8	0.369	0.416	0.548	0.220	0.310
-57°	21.3	11.6	0.284	0.351	0.557	0.210	0.319
-66°	17.6	17.6	0.424	0.445	0.643	0.258	0.382
-39°	7.9	17.0	0.381	0.389	0.576	0.228	0.319
51°	9.1	15.8	0.279	0.360	0.546	0.209	0.308
-35°	12.8	14.0	0.285	0.366	0.654	0.213	0.354
35°	11.6	10.9	0.338	0.407	0.596	0.242	0.342
66°	16.4	18.2	0.327	0.411	0.640	0.245	0.382
39°	14.0	14.6	0.393	0.480	0.563	0.232	0.315
78°	18.8	19.5	0.344	0.363	0.557	0.220	0.332

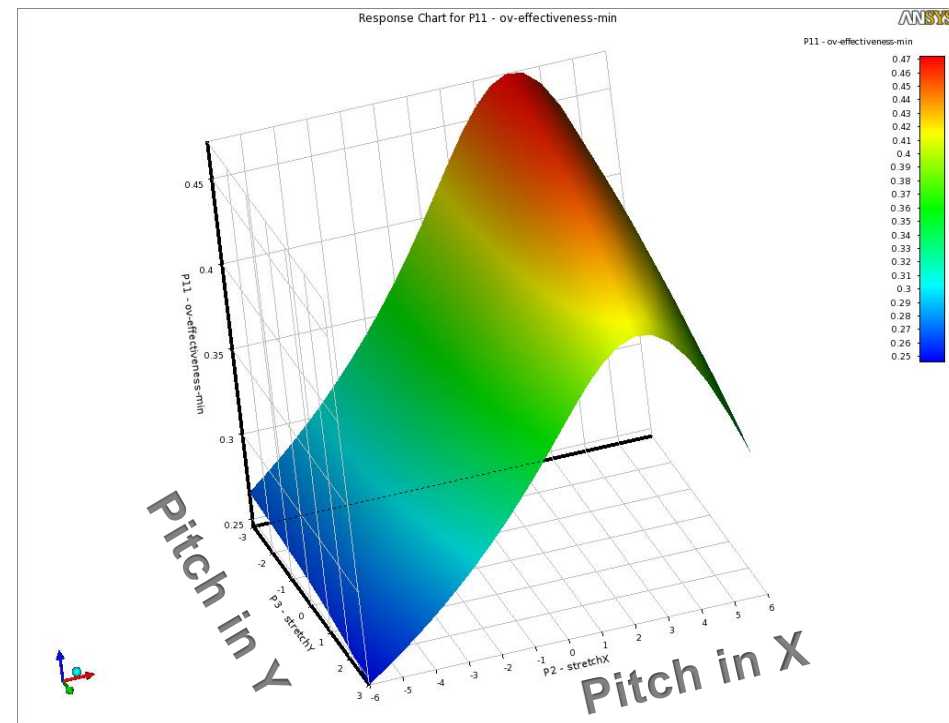
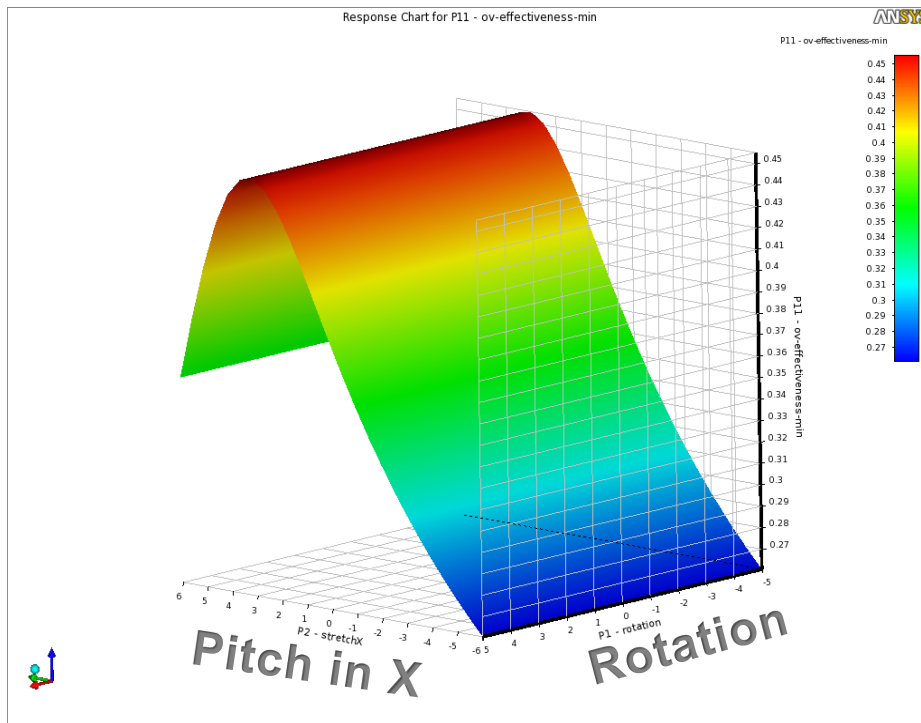
Answer surfaces

Optimization parameter
Overall effectiveness average = $f(\text{Input1}, \text{Input2})$



Answer surfaces

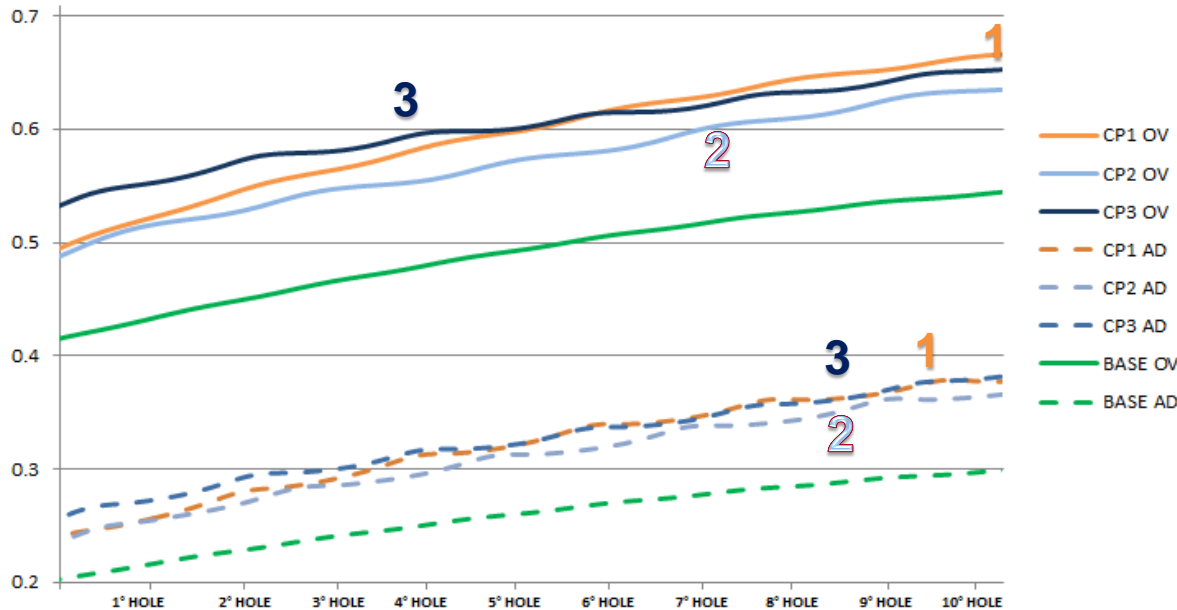
Constraint parameter $> 0,4$
Overall effectiveness min = $f(\text{Input1}, \text{Input2})$



Optimization Candidate Points



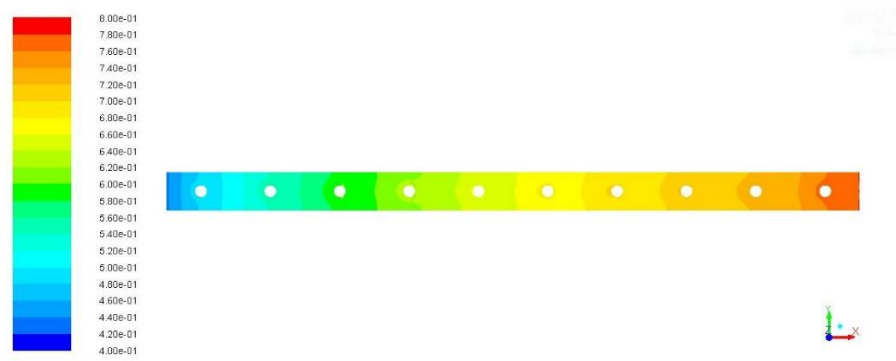
N°	Injection Angle (deg)	Pitch in x (mm)	Pirch in Y (mm)	Overall Effectiveness Min	Overall Effectiveness Average	Overall Effectiveness Max	Adiabatic Effectiveness Average	Adiabatic Effectiveness Max
BASE	90°	15,24	15,24	0.411	0.453	0.540	0.216	0.309
1	-32,7°	17,03	12,92	0.483	0.591	0.681	0.316	0.392
2	-33,2°	18,31	12,90	0.482	0.563	0.652	0.304	0.383
3	-74,6	16,72	12,90	0.524	0.603	0.668	0.338	0.403



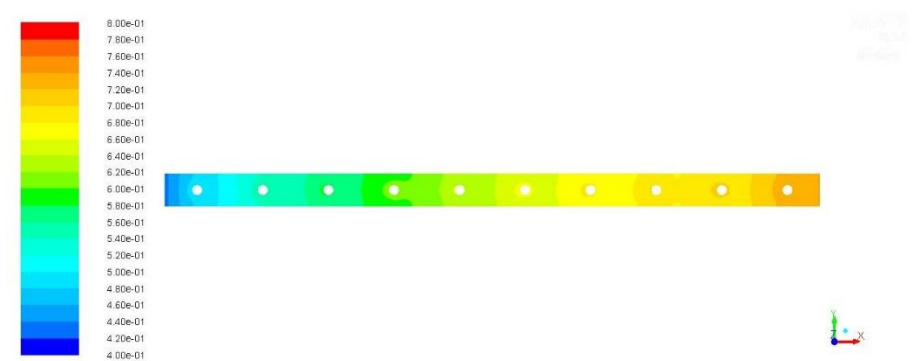
- Improvement of 30% in overall effectiveness
- Improvement of 50% in adiabatic effectiveness
- Candidate Point N°3 :
 - Overall Effectiveness higher
 - Lower Temperature gradient along the plate
 - Ad Effectiveness higher

Overall effectiveness on the Plate

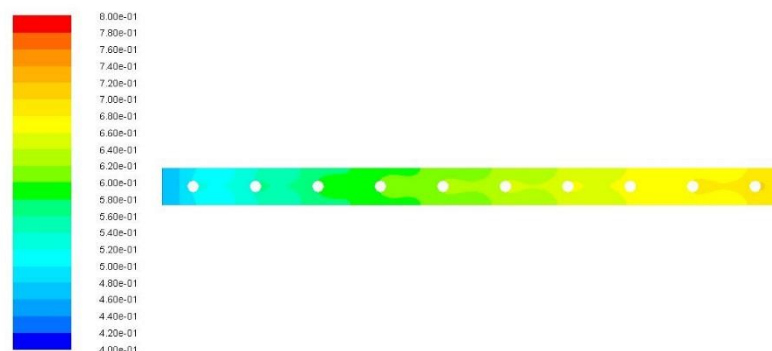
Candidate Point N°1



Candidate Point N°2



Candidate Point N°3

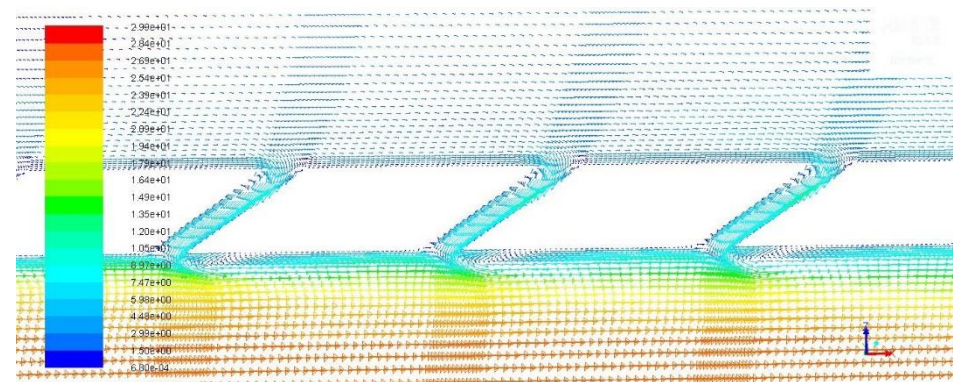
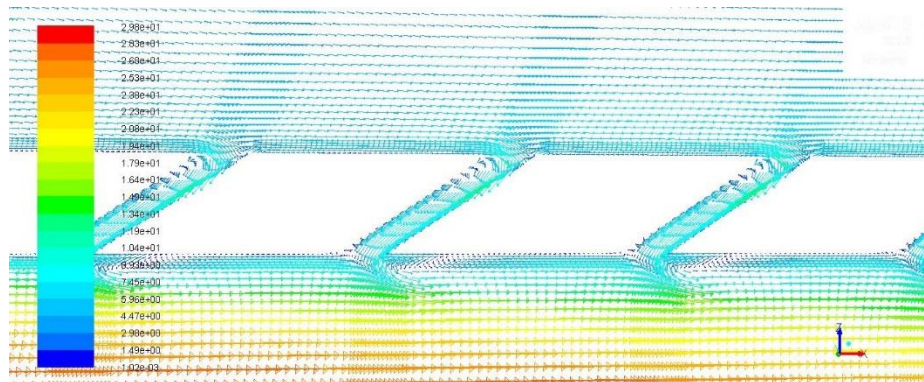


- **More homogeneous effectiveness**
- **Higher minimum effectiveness**

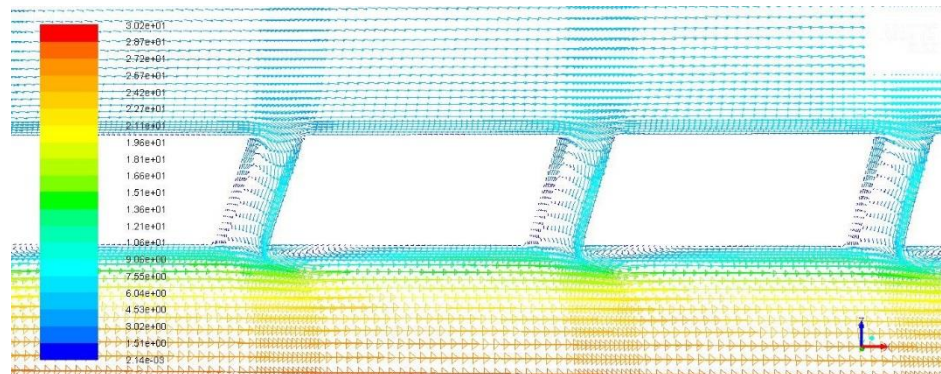
Velocity vectors on symmetry plane

Candidate Point N°1

Candidate Point N°2



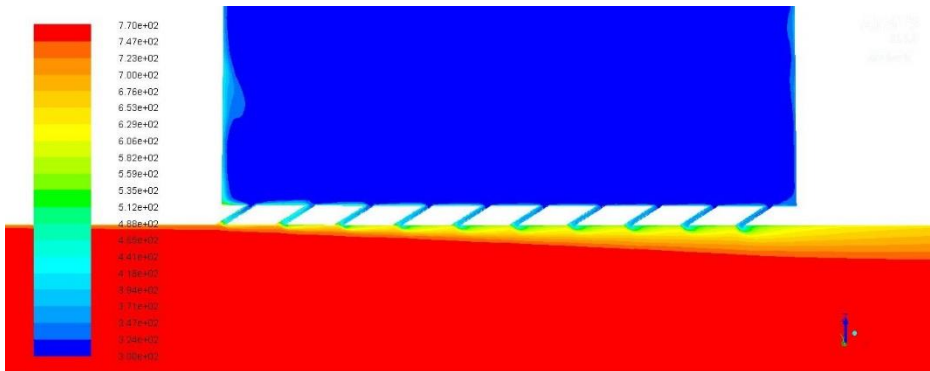
Candidate Point N°3



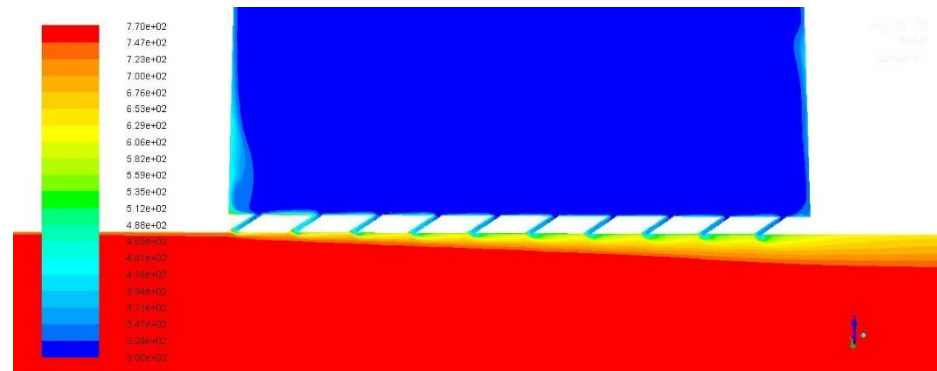
**Smaller
detachment and
recirculation zone**

Temperature profile on symmetry plane

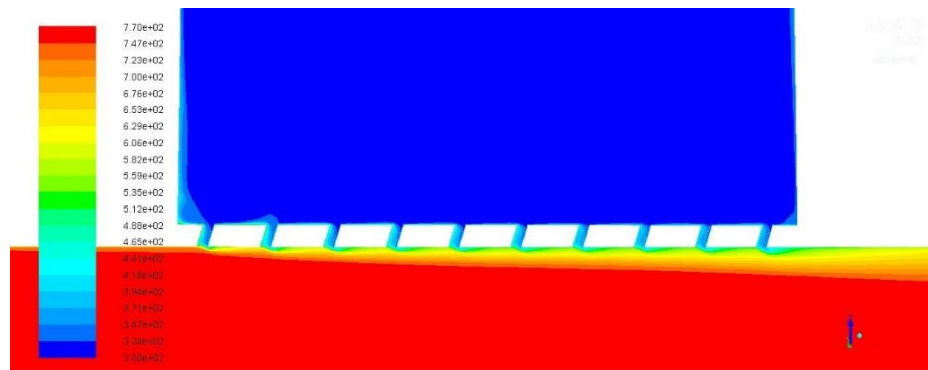
Candidate Point N°1



Candidate Point N°2



Candidate Point N°3

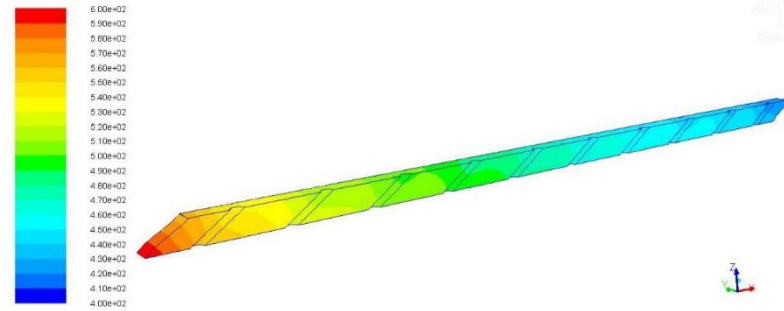
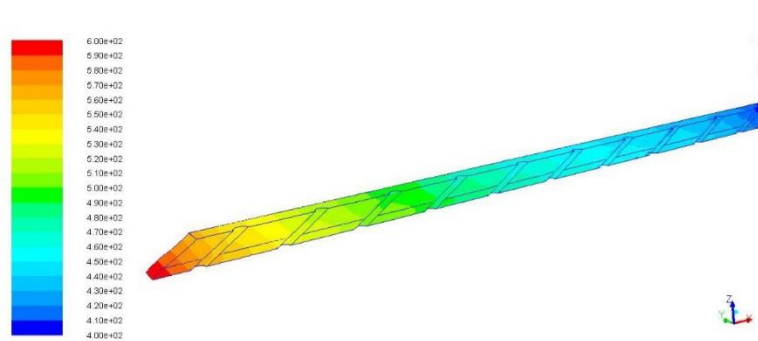


**Better Coverage
of the plate**

Temperature gradient on the plate

Candidate Point N°1

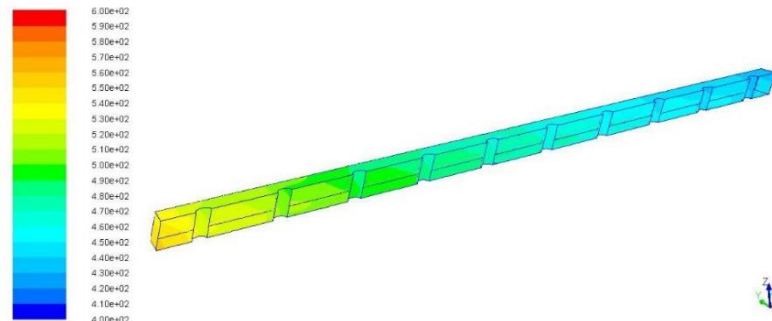
Candidate Point N°2



$$\nabla T_x \approx 200 \text{ K}$$

$$\nabla T_z \approx 30 \text{ K}$$

Candidate Point N°3

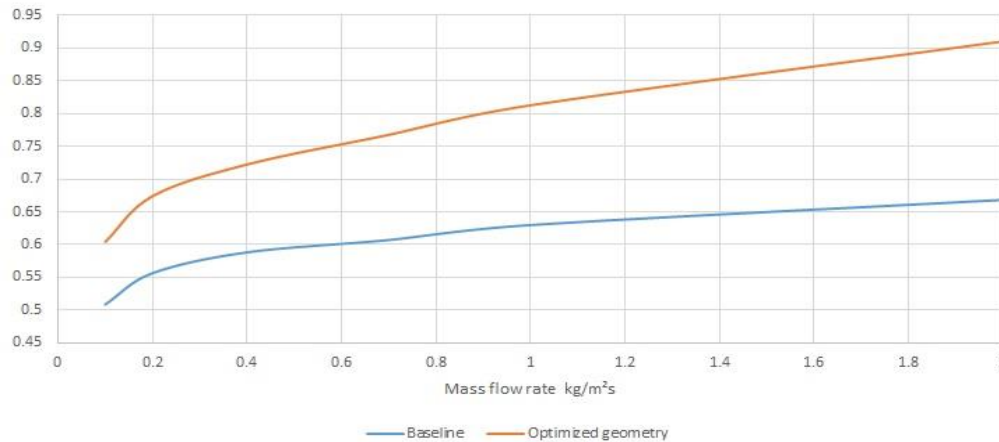


$$\nabla T_x < 130 \text{ K}$$

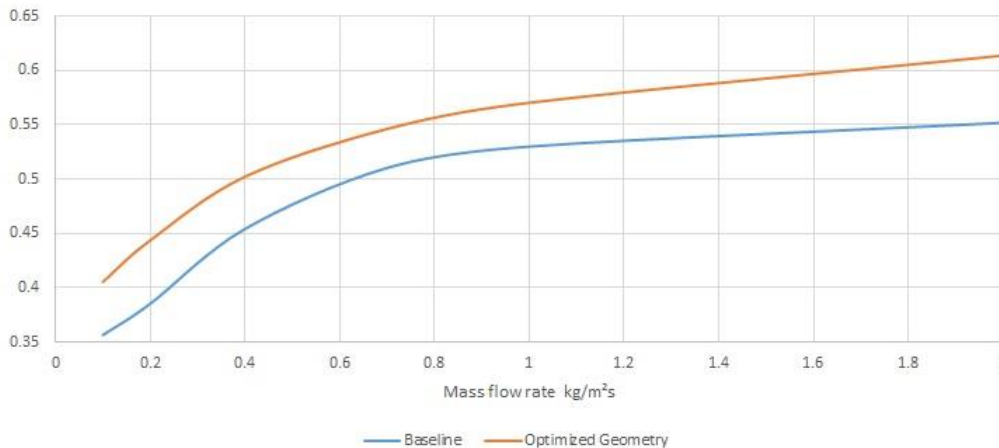
$$\nabla T_z < 10 \text{ K}$$

Overall effectiveness as a function of cooling air mass flow (G)

OVERALL EFFECTIVENESS MAX



OVERALL EFFECTIVENESS MIN



- New geometry is better than baseline for all G
- Get same effectiveness with lower G
- Plateaux for high G

- CFD numerical study of an effusion cooling system developed at ***University of Leeds***
- Model validation matching experimental data obtained from:
G E Andrews, A A Asere, M L Gupta and M C Mkpadi,
“Effusion cooling: the influence of the number of holes”
Proceedings of the Institution of Mechanical Engineers, Part A: Journal of Power and Energy 1990
- Shape optimization performed by means of Rbf Morph and Ansys Workbench suite
- Results analysis to get influence of the shape parameters on the effusion cooling effectiveness, improvement of 30%
- Found an optimal geometry reducing up to 10 times cooling air flow, without reducing effectiveness.



MONTREAL

ASME TURBO EXPO 2015 Palais des Congrès | June 15-19

TURBO EXPO

Turbine Technical Conference and Exposition

W. Savastano, A. Pranzitelli, G. E. Andrews, M. E. Biancolini, D. B. Ingham, M. Pourkashanian,

“Goal driven shape optimisation for conjugate heat transfer in an effusion cooling plate”,

Asme Turbo Expo, Montreal, Québec 2015



UNIVERSITY OF LEEDS

Thank you for your attention

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